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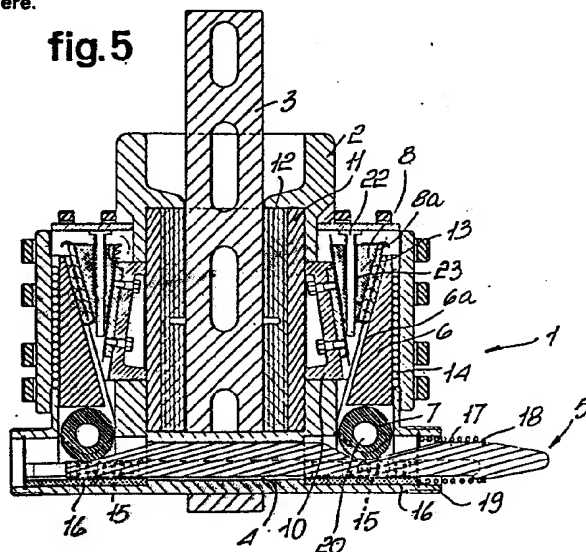
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84 Device for actuating disc brakes in motorvehicles or elsewhere.

87 This invention is concerned with a mechanism for actuating disc brakes in motorvehicles or elsewhere. The mechanism comprises a caliper (2) securely attached to the fixed main frame of a motorvehicle and carrying, on opposite sides with respect to a disc (3) of a disc brake, a pair of friction pads (12) which are engageable by contact with the opposed faces of the disc (3). The peculiar aspect of the mechanism is that it comprises a box-like body (1) securely attached to the cited caliper (2) and accommodating slidably therein a push rod (5) movable parallelly to the disc (3) axis; the push rod (5) has at least one ramp region which engages with a first wedge (6) having an inclined face (6a) mating in mutual sliding relationship with an inclined face (8a) of a self-adjustment wedge (8); the self-adjustment wedge (8) engages at its opposite face, relatively to the first wedge (6), with a plunger (10) acting on the friction pad (12) to urge it into engagement with the brake disc (3).

fig.5



"DEVICE FOR ACTUATING DISC BRAKES IN MOTORVEHICLES
OR ELSEWHERE"

This invention relates to a mechanism for actuating disc brakes in motorvehicles or elsewhere.

Known are several brake types, or devices adapted to produce a retarding action which opposes the motion
5 of a machine member or of a machine as a whole.

Barring electric brakes, which operate by converting kinetic energy into reusable electric power, the energy dissipated by any brake types is converted into heat.

Specifically, currently used disc brakes wherein
10 two friction pads are urged from opposite sides against the faces of a disc mounted rigid with moving parts, the amount of heat developed is considerable.

Such brake types, while providing a satisfactory braking effect, are not devoid of shortcomings. In fact,
15 owing to the friction pads being moved by hydraulic pistons, it is not unusual for the working liquid to partly turn into vapor, on account of the high temperatures generated, the compressible nature whereof results in the force applied to the pistons for braking
20 being cancelled or significantly decreased.

Another disadvantage of current disc brakes originates from the pressure seal rings of the hydraulic piston; in fact, their hardening or, even worse, their breaking results in oil leaking out and the operating
25 pressure level dropping below that required for the braking action, while the leaking oil may easily flow onto the pads and disc, thus jeopardizing the ability of the brake to develop friction.

The poor reliability of hydraulically operated

disc brakes makes their application to powerful and heavy industrial vehicles inadvisable.

It is the aim of this invention to obviate such prior drawbacks by providing a mechanism for actuating disc brakes, which drives the pistons acting on the friction pads through a fully mechanical drive arrangement, thus dispensing with the necessity for oil or other liquids under pressure in the direct proximities of the brake disc.

Within the above aim, it is an object of the invention to maintain in all situations a good transfer of the braking force to the pads acting on the brake discs.

This aim, as well as this and other objects such as will be apparent hereinafter, are achieved by a mechanism for actuating disc brakes in motorvehicles or elsewhere, comprising a caliper (2) rigidly associated with the fixed main frame of a motorvehicle and carrying on opposite sides with respect to a disc (3) of a disc brake a pair of friction pads (12) engageable by contact with the opposed faces of said disc (3), and characterized in that it comprises a box-like body (1) rigidly associated with said caliper (2) and accommodating slidably therein a push rod (5) movable parallelly to the axis of said disc (3), said push rod (5) having at least one ramp region (5a,5b) in engagement with a first wedge (6) having an inclined face (6a) in mutual sliding engagement with one inclined face (8a) of a self-adjustment wedge (8) adapted to engage, at the opposed face thereof with respect to said first wedge,

with a plunger (10) acting on a friction pad (12) to engage it with said brake disc (3).

Further features and advantages of the invention will be more clearly understood from the following
5 description of a preferred, but not exclusive, embodiment of the inventive mechanism, as illustrated by way of example and not of limitation in the accompanying drawings, where:

Figure 1 is a side elevation view of a brake disc
10 incorporating a mechanism according to the invention;

Figure 2 shows in side elevation and top plan views respectively, a detail of Figure 1;

Figure 3 shows in full view and in section another detail of a mechanism according to the invention;

15 Figure 4 shows two views of a further detail of the mechanism of this invention;

Figure 5 is a sectional view of Figure 1, taken along the line X-X;

Figure 6 shows in two views and a section the
20 self-adjustment wedge according to the invention;

Figure 7 shows in two views and a section the push rod according to the invention;

Figure 8 shows in two views and a section the plunger according to the invention; and

25 Figure 9 shows two views of the first wedge of this invention.

With reference to the drawing views, a device according to the invention comprises a box-like body,

generally designated with the reference numeral 1,
which is rigidly associated with a caliper 2 bonded
to the fixed main frame of a machine and extends above
both faces of a brake disc 3 rigid with a part to be
5 braked.

Formed on the interior of the box-like body is a
seat 4 which accommodates a push rod 5 slidably therein,
said push rod extending parallel to the axis of the
disc 3 and being movable relatively to the box-like
10 body in a substantially parallel direction to its own
axis.

The push rod 5 has two regions 5a and 5b,
respectively, of ramp-like configuration, each engaging
with a first wedge 6 from opposed sides with respect
15 to the brake disc. In order to minimize wear and
friction, between said ramp of the push rod 5 and the
first wedge 6, there is provided a roller 7 which is
associated rotatably with the first wedge 6.

Thus, the first wedge is made movable
20 perpendicularly to the axis of the push rod 5 upon the
rod being moved parallelly to its own axis.

The first wedge is formed with an inclined face
6a slidably mating with an inclined face 8a of a self-
adjustment wedge 8 movable in a substantially parallel
25 direction to the axis of the push rod 5.

On the side away from this inclined face, the
self-adjustment wedge engages with a plunger 10 which
engages, in turn, with a friction pad 12 through a
pad holder 11.

30 Again in order to reduce frictional resistance,

between the first wedge and self-adjustment wedge,
there are provided a first plurality of bearing needles
13, and a second plurality of bearing needles 14 are
provided between the first wedge and wall of the box-
like body.

To review the individual components in more detail,
it may be seen that the push rod is formed at the
bottom with a sliding surface 5c adapted to engage with
a plurality of shoes 15 carried on a sliding seat 16,
shown best in Figure 4, which is rigidly associated
with the walls of the seat 4.

The push rod 5 further carries, externally and
coaxially therewith, a biasing spring 17 having one end
engaged with a first ring 18 carried on the push rod 5
and the other end in abutment relationship with a
second ring 19 carried in the seat 4.

The first wedge 6 has on its push rod side a pair
of flanges 6b and 6c wherein a shaft 20 of the roller
7 engages rotatably.

The first wedge and self-adjustment wedge are
formed, on the mutually confronting inclined faces
thereof, with guide grooves 6d and 8c, respectively,
which accommodates the needles of said first plurality
of bearing needles.

The self-adjustment wedge has on its face away
from the face confronting said first wedge a plurality
of teeth 8b adapted to engage with a plurality of
teeth 21b correspondingly carried on a small plate 21,
shown best in Figure 3 and being rigidly associated
with the plunger 10.

Across the self-adjustment wedge, moreover,
there extends in a substantially perpendicular
direction to its direction of movement in operation
a guide seat 8d accommodating a rod 22 and self-
5 adjustment spring 23. The rod 22 abuts, under the bias
from the self-adjustment spring 23, against a wall of
the box-like body such as to urge the self-adjustment
wedge to slide against the plunger. Thus, automatic
adjustment of the mechanism is achieved as the friction
10 pads wears, since the tooth pluralities only permit
for movement of the self-adjustment wedge in one
direction.

For completeness of illustration, it should be
added that the box-like body is provided with a
15 reaction plug 24, shown best in Figure 2, which defines,
with its inside surface, a guide sliding seat 24a in
the box-like body for the first wedge. As mentioned,
in order to reduce friction between the first wedge and
this surface, there are provided a second plurality of
20 bearing needles 14.

The operation of the inventive mechanism may be
appreciated from the foregoing description.

On applying the brake, the push rod is driven
along its own axis, either mechanically or hydraulic-
25 ally, and caused to move, through the roller 7, the
first wedge 6 which becomes operative to shift the
self-adjustment wedge and, accordingly, the plunger
which urges, in turn, the friction pad against the
brake disc.

30 It has been found that, in actual practice, a

mechanism as provided by the invention achieves, with wedges ensuring that the drive takes place through ramp surfaces, an increase of the actuating force which contributes to the reliability of the braking system
5 as a whole.

Furthermore, where the push rod is actuated hydraulically, the working liquid is kept away from the contact area of the pads with the disc, and hence away from areas liable to attain high temperatures.

10 In the course of actual tests, it has been found that best results, as regards drive, increasing actuating force and operation, are to be achieved with the inclined faces of the push rod and wedges set at an inclination angle of 15°.

15 The above description makes reference to a mechanism acting on both faces of the brake disc, but it should be understood that, similarly to other braking systems for automobiles, this mechanism could also use a single operational unit and the caliper
20 left floating.

It should be further added that, owing to the simple form of actuation provided, the inventive mechanism may be implemented in modular assemblies to be mounted in banks for simultaneously braking
25 several brake discs through the actuation of a single push rod arranged to act against all of the first wedges in the braking arrangement.

The mechanism as disclosed is susceptible to many modifications and changes without departing from
30 the scope of this inventive idea; further, all of the

details may be replaced with technically equivalent elements.

In practicing the invention, the materials used, as well as the dimensions, may be any ones contingent
5 on individual requirements and the state of the art.

CLAIMS

1 1. A mechanism for actuating disc brakes in
2 motorvehicles or elsewhere, comprising a caliper
3 rigidly associated with the fixed main frame of a
4 motorvehicle and carrying on opposite sides with
5 respect to a disc of a disc brake a pair of friction
6 pads engageable by contact with the opposed faces of
7 said disc, and characterized in that it comprises a
8 box-like body rigidly associated with said caliper
9 and accommodating slidably therein a push rod movable
10 parallely to the axis of said disc, said push rod
11 having at least one ramp region in engagement with a
12 first wedge having an inclined face in mutual sliding
13 engagement with one inclined face of a self-adjustment
14 wedge adapted to engage, at the opposed face thereof
15 with respect to said first wedge, with a plunger acting
16 on a friction pad to engage it with said brake disc.

1 2. A mechanism according to Claim 1, characterized
2 in that said push rod has two ramp-like regions each
3 in engagement with a first wedge on opposed sides with
4 respect to said disc.

1 3. A mechanism according to Claims 1 and 2,
2 characterized in that it comprises at least one roller
3 interposed between each of said ramp-like regions of
4 said push rod and said first wedge.

1 4. A mechanism according to the preceding claims,
2 characterized in that it comprises a first plurality
3 of bearing needles interposed between said first wedge
4 and said self-adjustment wedge.

1 5. A mechanism according to one or more of the

2 preceding claims, characterized in that said self-
3 adjustment wedge has on the face thereof in engagement
4 with said plunger a plurality of teeth meshing with
5 a plurality of teeth carried on said plunger, said self-
6 adjustment wedge being slidably movable relatively to said
7 plunger under the action of a biasing means in the
8 home position to take up the wear of said friction pad.

1 6. A mechanism according to one or more of the
2 preceding claims, characterized in that said box-like
3 body comprises a reaction plug defining, on the inside
4 surface thereof, a guide sliding seat for said first
5 wedge.

1 7. A mechanism according to one or more of the
2 preceding claims, characterized in that said first
3 wedge is slidably movable relatively to said reaction
4 plug, a second bearing needle plurality being provided
5 between said guide sliding seat and said first wedge.

1 8. A mechanism according to one or more of the
2 preceding claims, characterized in that said push rod
3 is slidably movable relatively to said box-like body
4 in a substantially parallel direction to its own axis,
5 a plurality of shoes being interposed between said rod
6 and the seat accommodating it inside said box-like
7 body.

1 9. A mechanism according to one or more of the
2 preceding claims, characterized in that said first wedge
3 is pivotally associated with said roller intervening
4 between said first wedge and said push rod.

fig. 1

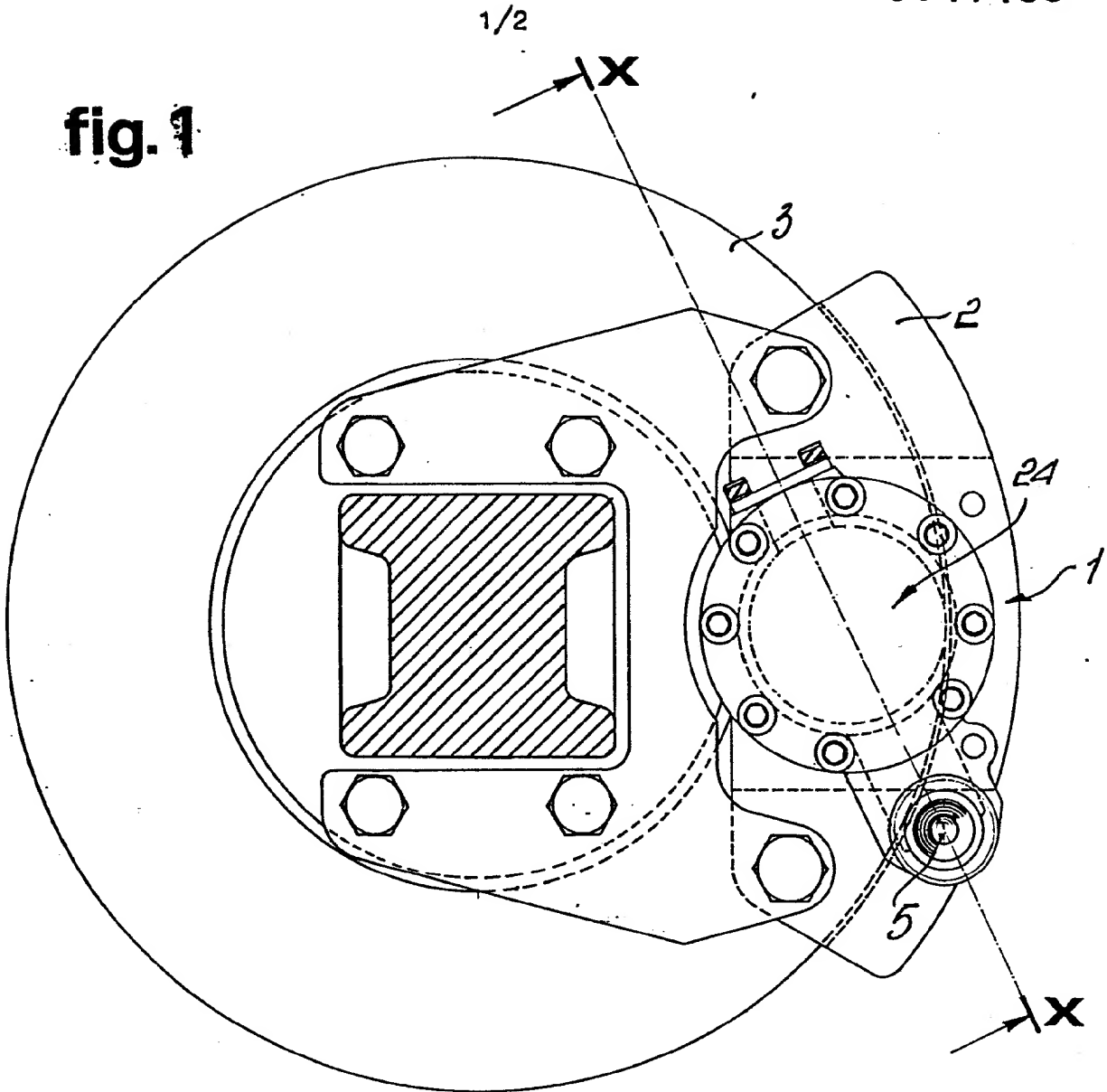


fig. 2

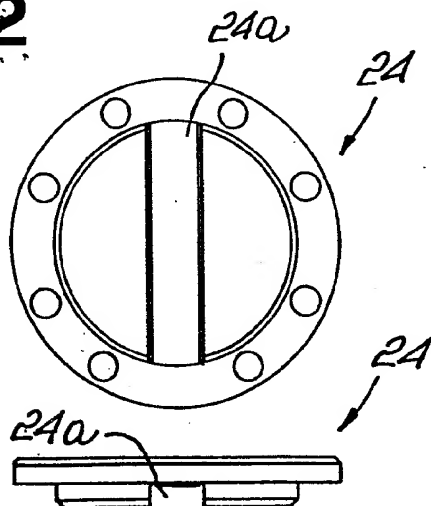


fig. 3

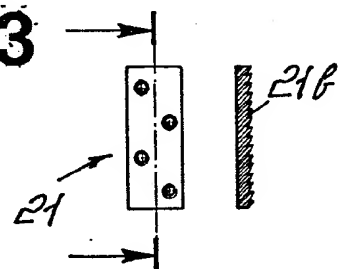


fig. 4



fig. 5

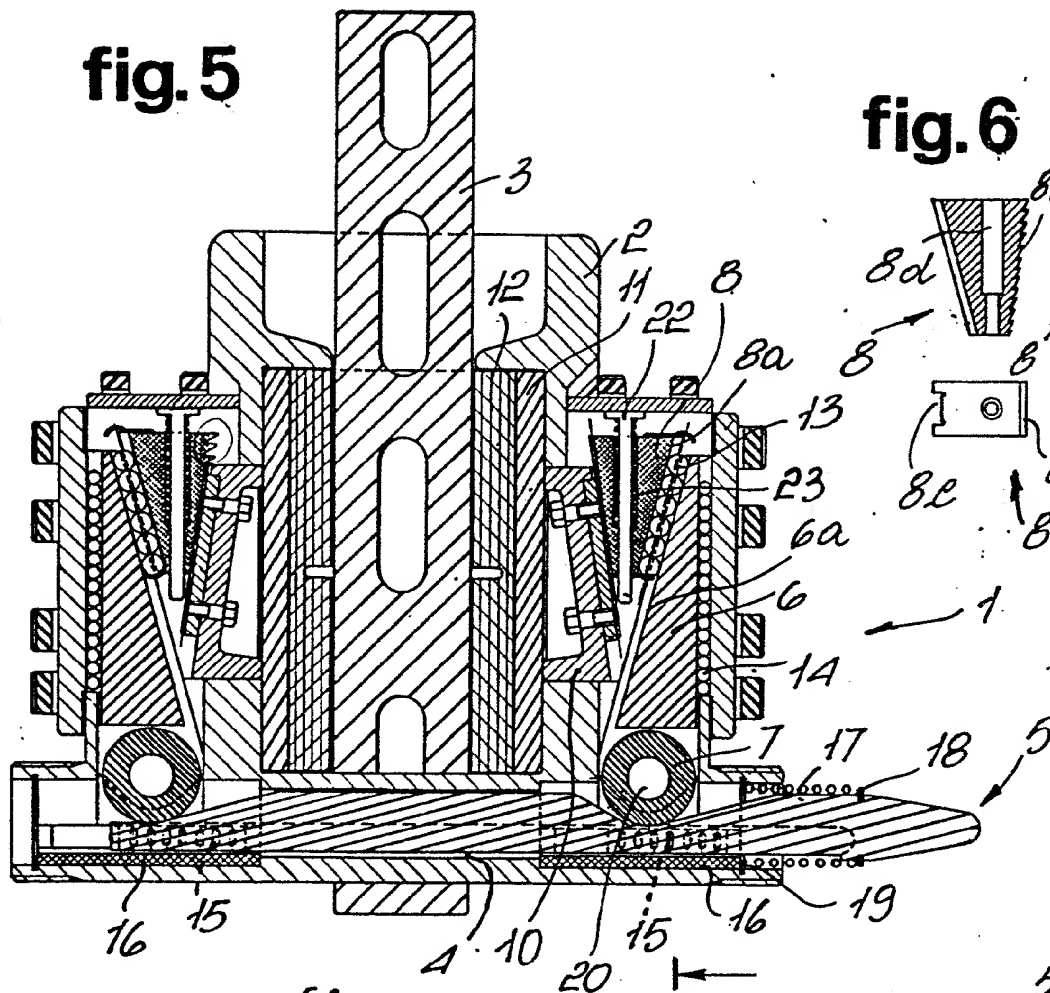


fig. 6

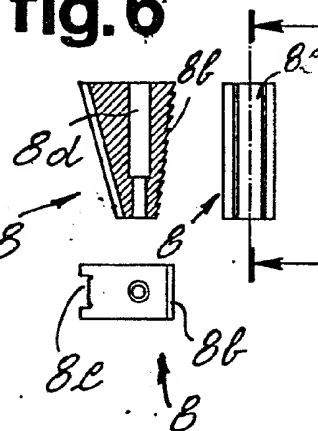


fig. 7

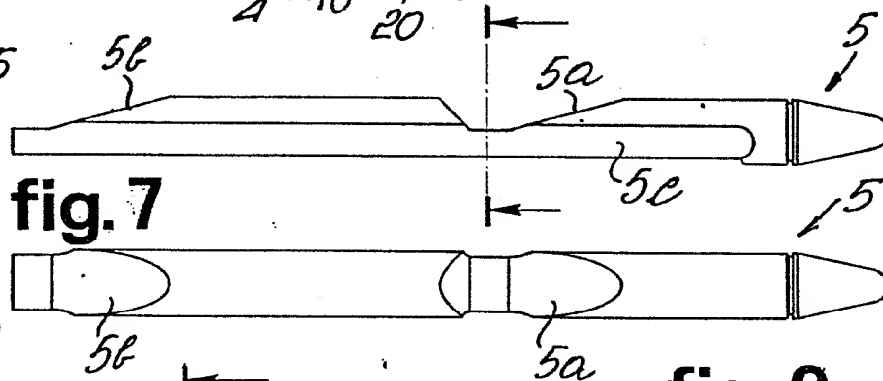


fig. 8

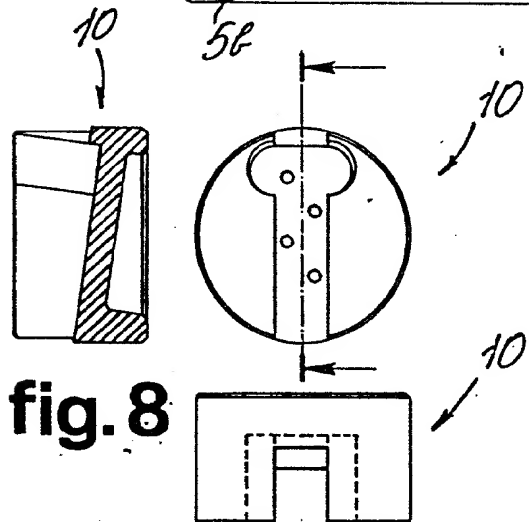
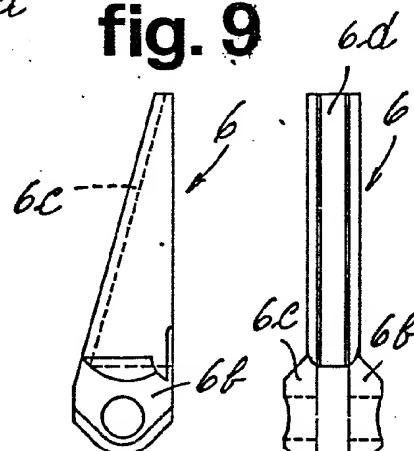


fig. 9





European Patent
Office

EUROPEAN SEARCH REPORT

0141406

Application number

EP 84 11 3189

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int. Cl.4)
A	FR-A-1 345 576 (REGIE NATIONALE DES USINES RENAULT) * Whole document, figures 1-3 *	1	F 16 D 65/54
A	EP-A-0 002 632 (VERLINDE, SOCIÉTÉ ANONYME) * Whole document, figures 1,2 *	1	
A	DE-A-1 675 155 (LICENTIA PATENT-VERWALTUNGS-GmbH) * Whole document, figure *	1	
A	DE-A-2 609 465 (WABCO WESTINGHOUSE AIR BRAKE CO.)		TECHNICAL FIELDS SEARCHED (Int. Cl.4)
A	DE-A-2 151 192 (AIRHEART PRODUCTS INC.)		F 16 D 65/00 B 61 H 15/00
The present search report has been drawn up for all claims			
Place of search THE HAGUE		Date of completion of the search 14-02-1985	Examiner BRAEMS
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